Amendments to the Claims:

[1] (Currently Amended) A method of controlling pressure in an electric injection molding machine, comprising:

detecting an angular velocity ω of a motor operative to propel forward a screw in an injection molding machine;

deriving an estimated melt pressure value δ^* without deriving a differential of the detected angular velocity ω , based on an observer, from said detected angular velocity ω of said motor and a torque command value T^{CMD} given to said motor; and

controlling said motor such that said estimated melt pressure value δ^* follows a mult pressure setting δ^{REF} .

[2] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1, wherein said observer is represented by the following Expression 1.

[Expression 1]

where on': Estimated value of Augular velocity of Motor

 d_1 , d_2 : Certain coefficients

- J: Inertia moment over Injection mechanism
- $F(\omega)$: Dynamic frictional resistance and Static frictional resistance over Injection mechanism
- [3] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1, wherein said observer is represented by the following Expression 2.

$$\begin{split} \omega^* = \omega^*_{-1} + \{ d_1(\omega^*_{-1} - \omega) + (1/J) \ (T^{CMD}_{-1} + \delta^*_{-1} + F(\omega)) \} \ d \ t \\ \delta^* = \delta^*_{-1} + \{ d_2(\omega^*_{-1} - \omega) \} \ d \ t \\ [\text{Expression 2}] \end{split}$$

where ω^: Estimated value of Angular velocity of Motor

- d₁, d₂: Certain coefficients
- J: Inertia moment over Injection mechanism
- $F(\omega)$: Dynamic frictional resistance and Static frictional resistance over Injection mechanism
- \mathbf{x}_{\perp} : Value of \mathbf{x} at Immediately preceding processing period
- [4] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1,

wherein said screw in said injection molding machine and said motor are coupled together via a belt suspended around pulleys mounted on respective rotation shafts, and wherein said observer is represented by the following Expression 3.

[Expression 3]

$$\frac{d}{dt} \begin{bmatrix} \hat{\omega}^{M} \\ \hat{\omega}^{l} \\ \hat{E} \\ \hat{S} \\ \hat{\sigma}^{l} \end{bmatrix} = \begin{bmatrix} d_{1} & 0 & -\frac{R^{M}}{J^{M}} & 0 & 0 \\ d_{2} & 0 & \frac{R^{L}}{J^{L}} & 1 & 0 \\ d_{3} & 0 & \frac{R^{L}}{J^{L}} & 1 & 0 \\ d_{1} & K_{\infty} & K_{\infty} \frac{R^{L}}{J^{L}} & K_{\infty} & 1 \\ d_{3} & 0 & 0 & 0 \\ d_{1} & 0 & 0 & 0 & 0 \\ d_{3} & 0 & 0 & 0 & 0 \\ \end{bmatrix} + \begin{bmatrix} 1 \\ \hat{\omega}^{N} \\ \hat{\omega}^{l} \\ \hat{E} \\ \hat{\sigma}^{l} \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} F^{CMB} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ K_{\infty} \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} d_{1} \\ d_{2} \\ 0 \\ 0 \\ 0 \end{bmatrix} e^{MB} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{J^{L}} \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\$$

where di-ds: Certain coefficients

JM: Inertia moment at Motor side

ω^M: Angular velocity of Motor

 $\ensuremath{\mathbb{R}}^{\ensuremath{\mathsf{N}}}\colon$ Pulley radius at Motor side

F: Tension of Belt

Kb: Spring constant of Belt

Jh: Inertia moment at Screw side

ω": Angular velocity at Screw side

Rh: Pulley radius at Screw side

 $F_{\mathbf{d}}\left(\boldsymbol{\sigma}^{b}\right):$ Dynamic frictional resistance at Screw side

 K_w : Elastic modulus of Resin

Ked: Coefficient of Viscosity of Resin

σ: Force of Screw pushing Resin

[5] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1,

wherein said screw in said injection molding machine and said motorage coupled together via a belt suspended around pulleys mounted on respective rotation shafts, and

wherein said observer is represented by the following $Expression \ 4.$

[Expression 4]

$$\begin{split} & \tilde{\boldsymbol{\omega}}^{M} = \tilde{\boldsymbol{\omega}}^{M-1} + \left\{ d_{+} \left(\hat{\boldsymbol{\omega}}^{M-1} - \boldsymbol{\omega}^{M} \right) + \frac{1}{J^{M}} \left(\boldsymbol{r}^{\text{CSM}} - \boldsymbol{1} - \boldsymbol{R}^{M} \, \hat{\boldsymbol{F}}_{-1} \right) \right\} dt \\ & \tilde{\boldsymbol{\omega}}^{L} = \tilde{\boldsymbol{\omega}}^{L} + \left\{ d_{+} \left(\hat{\boldsymbol{\omega}}^{M} - \boldsymbol{1} - \boldsymbol{\omega}^{M} \right) + \frac{1}{J^{L}} \left(\boldsymbol{R}^{L} \hat{\boldsymbol{F}}_{-1} + \hat{\boldsymbol{F}}_{-1} + F_{d} \left(\boldsymbol{\omega}^{L} \right) \right) \right\} dt \\ & \tilde{\boldsymbol{F}} = \hat{\boldsymbol{F}}_{-1} + \left\{ d_{+} \left(\hat{\boldsymbol{\omega}}^{M} - \boldsymbol{\omega}^{M} \right) + K_{L} \left(\boldsymbol{R}^{M} \tilde{\boldsymbol{\omega}}^{M} - \boldsymbol{1} - \boldsymbol{R}^{L} \tilde{\boldsymbol{\omega}}^{L} - \boldsymbol{L}^{L} \right) \right\} dt \\ & \tilde{\boldsymbol{S}} = \hat{\boldsymbol{\delta}}_{-1} + \left\{ d_{+} \left(\hat{\boldsymbol{\omega}}^{M} - \boldsymbol{\omega}^{M} \right) + K_{u} \tilde{\boldsymbol{\omega}}^{L} + \frac{K_{u} \tilde{\boldsymbol{\omega}}^{L} - K_{u}^{L} + \hat{\boldsymbol{F}}_{-1} + \hat{\boldsymbol{F}}_{d} \left(\boldsymbol{\omega}^{L} \right) \right) + \tilde{\boldsymbol{\sigma}}_{-1} \right\} dt \\ & \tilde{\boldsymbol{\sigma}} = \tilde{\boldsymbol{\sigma}}_{-1} + d_{+} \left(\hat{\boldsymbol{\omega}}^{M} - \boldsymbol{\omega}^{M} \right) dt \end{split}$$

where di-do: Certain coefficients

 J^{M} : Inertia moment at Motor side

 ω^M : Angular velocity of Motor

R^M: Pulley radius at Motor side

F: Tension of Bell

K_b: Spring constant of Belt

J^h: Inertia moment at Screw side

ω^L: Angular velocity at Screw side

R^L: Pulley radius at Screw side

 $F_{\sigma}(\omega^{h})$: Dynamic frictional resistance at Screw side

 K_{ω} : Elastic modulus of Resin

Kwd: Coefficient of Viscosity of Resin

σ: Force of Screw pushing Resin

x 1: Value of x at Immediately preceding processing period

[6] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1,

wherein said screw in said injection molding machine and said motor are coupled together via a belt suspended around pulleys mounted on respective rotation shafts, and

wherein said observer is represented by the following $\ensuremath{\mathtt{Expression}}$ 5.

[Expression 5]

$$d \begin{bmatrix} \hat{\omega}^{u} \\ \hat{\omega}^{l} \\ \hat{F} \\ \hat{\delta} \end{bmatrix} = \begin{pmatrix} d_{1} & 0 & -\frac{R^{u}}{\hat{J}^{u}} & 0 \\ d_{2} & 0 & \frac{R^{u}}{\hat{J}^{u}} & \frac{1}{\hat{J}^{u}} \\ d_{3} & 0 & \frac{R^{u}}{\hat{J}^{u}} & \frac{1}{\hat{J}^{u}} \\ d_{3} + K_{h}R^{u} & -K_{h}R^{u} & 0 & 0 \end{pmatrix} \begin{pmatrix} \hat{\omega}^{u} \\ \hat{\omega}^{l} \\ \hat{F} \\ \hat{\delta} \end{pmatrix} \mathbf{1} \begin{pmatrix} \frac{1}{\hat{J}^{u}} \\ 0 \\ 0 \\ 0 \end{pmatrix} \mathcal{T}^{CMD} + \begin{pmatrix} 0 \\ \frac{1}{\hat{J}^{u}} \\ 0 \\ 0 \end{pmatrix} \mathcal{F}_{d}(\omega^{u}) - \begin{pmatrix} d_{1} \\ d_{2} \\ d_{3} \\ d_{4} \end{pmatrix} \omega^{u}$$

where d₄ d₄: Certain coefficients

 $\boldsymbol{J}^{N}\colon$ Inertia moment at Motor side

 $\omega^M\colon \operatorname{Augular}$ velocity of Motor

RM: Pulley radius at Motor side

F: Tension of Belt

Kb: Spring constant of Belt

J^b: Inertia moment at Screw side

ω^L: Angular velocity at Screw side

R^L: Pulley radius at Screw side

 $F_d(\omega^b)$: Dynamic frictional resistance at Screw side

[7] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1,

wherein said screw in said injection molding machine and said motor are coupled together via a belt suspended around pulleys mounted on respective rotation shafts, and

wherein said observer is represented by the following

[Expression 6]

$$\begin{split} &\dot{\omega}^{M} = \dot{\omega}^{M}_{-1} + \left\{ d_{1} \left(\dot{\tilde{\omega}}^{M}_{-1} - \omega^{M} \right) + \frac{1}{J^{M}} \left(T^{CMD}_{-1} - R^{M} \hat{F}_{-1} \right) \right\} dt \\ &\dot{\omega}^{L} = \dot{\omega}^{L}_{-1} + \left\{ d_{2} \left(\dot{\tilde{\omega}}^{M}_{-1} - \omega^{M} \right) + \frac{1}{J^{L}} \left(R^{L} \hat{F}_{-1} + \tilde{F}_{a} + F_{a} \left(\omega^{L} \right) \right) \right\} dt \\ &\dot{\tilde{F}} = \hat{F}_{-1} + \left\{ d_{3} \left(\dot{\tilde{\omega}}^{M}_{-1} - \omega^{M} \right) + K_{b} \left(R^{M} \dot{\tilde{\omega}}^{M}_{-1} - R^{L} \dot{\tilde{\omega}}^{L}_{-1} \right) \right\} dt \\ &\dot{\tilde{S}} = \tilde{\delta}_{-1} + d_{4} \left(\dot{\tilde{\omega}}^{M}_{-1} - \omega^{M} \right) dt \end{split}$$

where d₁-d₄: Certain coefficients

JM: Inertia moment at Motor side

 $\boldsymbol{\omega}^{N} \colon$ Angular velocity of Motor

 $\mathbb{R}^{\mathbb{N}}$: Pulley radius at Motor side

F: Tension of Belt

Kb: Spring constant of Belt

Jh: Inertia moment at Screw side

ω": Angular velocity at Screw side

Rh: Pulley radius at Screw side

 $F_d(\omega^L)$: Dynamic frictional resistance at Screw side

x 1: Value of x at Immediately preceding processing period

[8] (Original) The method of controlling pressure in an electric injection molding machine according to claim 3, 5 or 7, further comprising:

calculating said torque command value $\mathfrak{T}^{\text{exp}}$ for said motor based the following Expression T_i and

feeding back said torque command value to said motor.

$$T^{CMD} = k p (\delta^{REF} - \delta^{A}) + \alpha$$
 [Expression 7]

where kp: Certain constant

α: Certain function or constant

[9] (Currently Amended) An apparatus for controlling pressure in an electric injection molding machine, comprising:

an observer arithmetic unit operative to derive an estimated melt pressure value δ^{*} without deriving a differential of the detected angular velocity ω , based on an observer, from an angular velocity ω of a motor operative to propel forward a screw in an injection molding machine and a torque command value T^{CMO} given to said motor; and

a torque arithmetic unit operative to calculate said torque command value T^{CRD} for said motor using said estimated melt pressure value δ^{*} derived at said observer arithmetic unit and feed back said torque command value to said motor.

[10] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1, further comprising deriving a dynamic frictional resistance F(w) from a relation between a velocity or position and a torque or current value associated with said motor at the time of injection with no resin loaded.

[11] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1, further comprising:

defining a dynamic frictional resistance F(0) as a sum of a velocity-dependent component and a load-dependent component;

deriving said velocity-dependent component of said dynamic frictional resistance from a relation between a velocity or position and a torque or current value associated with said motor at the time of air shot; and

deriving said load-dependent component of said dynamic frictional resistance from a relation between a torque or current value and a pressure value at the time of injection with a plugged nozzle.